

Port-Hamiltonian Systems for Fluid & Structural mechanics: Part I.

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ANR Project HAMECMOCPSYS on "*Hamiltonian Methods for the Control of Multidomain Distributed Parameter Systems*"

— Friday, May 22nd, 2015 —

Toulouse, France.

a 1-d.o.f. oscillator

Original dynamics: $m\ddot{x} + kx = 0$, with (x_0, \dot{x}_0) initial data, (1)

usually rewritten in state-space form, as:

$$\frac{d}{dt} \begin{bmatrix} x \\ \dot{x} \end{bmatrix} = \begin{bmatrix} 0 & \frac{1}{m} \\ -k & 0 \end{bmatrix} \begin{bmatrix} x \\ \dot{x} \end{bmatrix}, \quad \text{with } \begin{bmatrix} x_0 \\ \dot{x}_0 \end{bmatrix} \text{ initial data.} \quad (2)$$

⇒ Choose the **Hamiltonian formalism!**

- **Energy variables:** position $q := x$, momentum $p := m\dot{x}$,

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- **Co-energy variables:** $\partial_q H = kq$ force, $\partial_p H = \frac{1}{m}p := \dot{x}$ velocity,
- **Dynamical system:**

$$\frac{d}{dt} X = \begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix} \begin{bmatrix} kq \\ \frac{1}{m}p \end{bmatrix} = J \mathbf{grad}_X H(X).$$

with **skew-symmetric** matrix $J := \begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix}$, i.e. $J^T = -J$.

a 1-d.o.f. oscillator (2)

Theorem

J *skew-symmetric* $\implies \frac{d}{dt}H(X(t)) = 0$.

Hence, the dynamical system is *conservative*, i.e. $H(X(t)) = H(X_0)$.

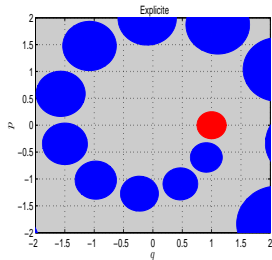
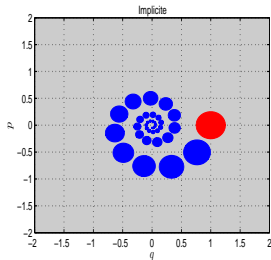
$$\begin{aligned} \frac{dH}{dt}(t) &= (\mathbf{grad}_X H(X), \frac{d}{dt}X)_{\mathbb{R}^2} \\ &= (\mathbf{grad}_X H(X), J \mathbf{grad}_X H(X))_{\mathbb{R}^2} \\ &= 0, \quad \text{since } J \text{ is skew symmetric!} \end{aligned}$$



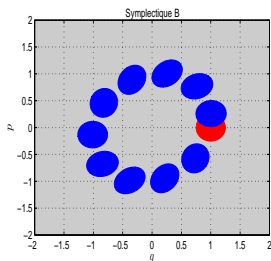
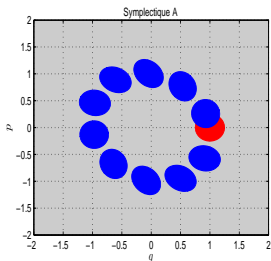
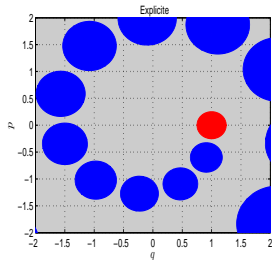
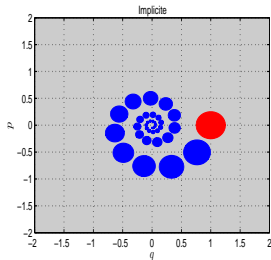
Severe consequences on numerical simulation!

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General ideas on Port Hamiltonian Systems (pHs)

- 1 strongly structured mathematical dynamical systems: both linear and non-linear, both finite-dimensional and infinite-dimensional,
- 2 based on physical grounds, allowing for different modelling levels,
- 3 all physics permitted: solid mechanics, structural mechanics, fluid mechanics, electromagnetism, electrical circuits, ...
- 4 comes along with specific numerical methods, which do preserve, at the discrete level, the structure of the continuous equations,¹
- 5 allows for open dynamical systems, with interacting ports,
- 6 modularity: interconnection of sub-systems, and... easy multiphysics modelling, e.g. Fluid-Structure Interaction²,
- 7 physically-based strategy for control and stabilization,
- 8 extensions to dissipative dynamical systems are available.

¹come on June, 18th, to ROMA Seminar by Saïd Aoues!

²wait till... Part II, by Flávio Ribeiro, in a few minutes!

Outline

- 1 Finite-dimensional case: Ordinary Differential Equations (ODEs)**
 - Useful Tools
 - Closed Systems
 - Open Systems
 - A short word on dissipation?
- 2 Infinite-dimensional case: Partial Differential Equations (PDEs)**
 - New Useful Tools
 - Closed systems
 - More examples in fluid mechanics!
 - A short word on dissipation? Navier-Stokes, at last!
- 3 Sloshing? a typical Fluid-Structure Interaction in Aeronautics!**

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Useful Tools in Finite Dimension

These 3 tools will be of major help in the study:

- the **gradient**: to be able to compute $\mathit{grad}_X H(X)$, when $X \in \mathbb{R}^{2n}$,
- **skew symmetric** matrices: $J^T = -J$,
- in the special case of quadratic Hamiltonian, hence linear dynamical systems $\dot{X} = AX$: **matrix exponentials**.

Straightforward consequences on **stability**:

- 1 $\text{spec}(A) \in \mathbb{C}_0^- \implies (\exp(tA) \rightarrow 0, \text{ as } t \rightarrow \infty)$, i.e. **asymptotic stability**.
- 2 $\exists \lambda \in \text{spec}(A), \Re(\lambda) > 0 \implies (\|\exp(tA)\| \rightarrow \infty, \text{ as } t \rightarrow \infty)$, i.e. (exponential) **instability**.
- 3 the case with eigenvalues on $i\mathbb{R}$ is more subtle, and requires geometric insight on the eigenspaces to be solved: either **stability** or (polynomial) **instability** can be found.

Ex 1: the n -d.o.f. linear oscillator

Original dynamics: $M\ddot{x} + Kx = 0$, with (x_0, \dot{x}_0) initial data, (3)

with mechanical parameters $M = M^T > 0$, $K = K^T \geq 0$.

- Energy variables: $q := x \in \mathbb{R}^n$, $p := M\dot{x} \in \mathbb{R}^n$, set $X = (q, p) \in \mathbb{R}^{2n}$,
- Hamiltonian function: $H(X) := \frac{1}{2}p^T M^{-1}p + \frac{1}{2}q^T Kq$,
- Co-energy variables: $\mathbf{grad}_q H = Kq$, and $\mathbf{grad}_p H = M^{-1}p := \dot{x}$,
- Dynamical system in standard form:

$$\frac{d}{dt}X = \begin{bmatrix} 0 & I \\ -I & 0 \end{bmatrix} \begin{bmatrix} Kq \\ M^{-1}p \end{bmatrix} = J \mathbf{grad}_X H(X),$$

with $J := \begin{bmatrix} 0 & I \\ -I & 0 \end{bmatrix}$ is skew-symmetric in \mathbb{R}^{2n} .

Ex.2: a 1-d.o.f. nonlinear oscillator: the pendulum

Original dynamics: $J\ddot{\theta} + g \sin(\theta) = 0$, with $(\theta_0, \dot{\theta}_0)$ initial data. (4)

⇒ Choose the **Hamiltonian formalism!**

- **Energy variables:** position $q := \theta$, momentum $p := J\dot{\theta}$,
- **Hamiltonian function:** $H(X) := \frac{1}{2J}p^2 + g(1 - \cos(\theta))$,
- **Co-energy variables:** $\partial_q H = g \sin(\theta)$ torque, $\partial_p H = \frac{1}{J}p := \dot{\theta}$ angular velocity,
- **Dynamical system** with $X = (q, p)$:

$$\frac{d}{dt}X = \begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix} \begin{bmatrix} g \sin(\theta) \\ \dot{\theta} \end{bmatrix} = J \mathbf{grad}_X H(X).$$

Theorem

J *skew-symmetric* ⇒ $\frac{d}{dt}H(X(t)) = 0$. Hence, the **non-linear** system is *conservative*, i.e. $H(X(t)) = H(X_0)$, with **non-quadratic** Hamiltonian.

Open Systems, Ports, and Energy Balance (1)

Suppose we have **interaction** with the environment, by means of:

- actuators, with control $u \in \mathbb{R}^m$,
- sensors, with co-localized measurements or observations $y \in \mathbb{R}^m$,

then, the port-Hamiltonian system (pHs) is defined by:

$$\dot{X} = J \mathbf{grad}_X H(X) + g(X) u(t), \quad (5)$$

$$y(t) = g(X)^T \mathbf{grad}_X H(X). \quad (6)$$

Theorem

J *skew-symmetric* \implies the system is *lossless*. Indeed,

$$\frac{d}{dt} H(X(t)) = (y(t), u(t))_{\mathbb{R}^m}, \text{ or } H(X(t)) = H(X_0) + \int_0^t (y(\tau), u(\tau))_{\mathbb{R}^m} d\tau.$$

What about the linear / quadratic case?

Suppose the Hamiltonian function is quadratic $H(X) := \frac{1}{2}X^T Q X$, with $Q = Q^T > 0$, we then easily compute $\mathbf{grad}_X H(X) = QX$, and we can define the closed linear dynamical system:

$$\dot{X} = J Q X,$$

that is, the **matrix of the dynamics** reads $A := J Q$.

Let $B := g$ be the **control matrix** of size $n \times m$, then the open dynamical system is given by:

$$\dot{X} = J Q X + B u(t), \quad (7)$$

$$y(t) = B^T Q X. \quad (8)$$

that is, the $m \times n$ **observation matrix** reads $C := B^T Q$.

Ex 1: n -d.o.f. damped oscillator

Original dynamics: $M\ddot{x} + (C + G)\dot{x} + Kx = 0$, with (x_0, \dot{x}_0) initial data, (9)

with $M = M^T > 0$, $K = K^T \geq 0$ and $C = C^T$, $G = -G^T$.

- Energy variables: $q := x \in \mathbb{R}^n$, $p := M\dot{x} \in \mathbb{R}^n$,
- Hamiltonian function: $H(X) = \frac{1}{2}p^T M^{-1}p + \frac{1}{2}q^T Kq$,
- Dynamical system in standard form:

$$\frac{d}{dt}X = \begin{bmatrix} 0 & I \\ -I & -(G + C) \end{bmatrix} \text{grad}_X H(X) = (J - R) \text{grad}_X H(X),$$

$$\text{with } J := \begin{bmatrix} 0 & I \\ -I & -G \end{bmatrix}, \text{ and } R := \begin{bmatrix} 0 & 0 \\ 0 & C \end{bmatrix}.$$

\implies Examine the two terms separately :

- Role of the skew-symmetric G matrix, the **gyroscopic term**.
- **Damping** effect is ensured, provided that $C = C^T \geq 0$.

Ex 1: about the G matrix

- 1 This matrix is often not considered in modelling processes of damping! Why?

When $C = 0$, whatever $G = -G^T$, the system proves conservative:
 $\frac{d}{dt}H_0(X(t)) = 0!$
- 2 Is it a naive generalizations due to mathematicians? No!

Classical mechanical example: **Coriolis force** $f = \omega \wedge \dot{x}$, with rotational speed $\omega = (p, q, r)^T$, $f = G_\omega \dot{x}$ where

$$G_\omega := \begin{bmatrix} 0 & -r & q \\ r & 0 & -p \\ -q & p & 0 \end{bmatrix} \text{ is skew symmetric.}$$
- 3 Also in electromagnetics: **Lorentz force** in a uniform magnetic field.

Open Systems with Damping, Energy Balance (2)

The port-Hamiltonian system (pHs) is defined by:

$$\dot{X} = (J - R) \mathbf{grad}_X H(X) + g(X) u(t), \quad (10)$$

$$y(t) = g(X)^T \mathbf{grad}_X H(X). \quad (11)$$

Theorem

J *skew-symmetric* and R *positive* \implies the system is *passive*. Indeed,

$$\begin{aligned} \frac{d}{dt} H(X(t)) &= -(\mathbf{grad}_X H(X), R \mathbf{grad}_X H(X))_{\mathbb{R}^n} + (y(t), u(t))_{\mathbb{R}^m}, \\ &\leq (y(t), u(t))_{\mathbb{R}^m}. \end{aligned}$$

Hence, $H(X(t)) - H(X_0) \leq \int_0^t (y(\tau), u(\tau))_{\mathbb{R}^m} d\tau$.

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Useful Notions in Infinite Dimension

These notions will be of major help in the following:

- **functions** u , instead of vectors X ,
- an infinite-dimensional **Hilbert functional space** \mathcal{H} for functions, instead of a finite-dimensional Euclian vector space \mathbb{R}^{2n} ,
- a Hamiltonian **functional** H , defined on functions u , instead of a Hamiltonian function defined on vectors, e.g.:

$$\begin{aligned} H : \mathcal{H} &\rightarrow \mathbb{R} \\ u &\mapsto H(u) := \frac{1}{2} \int_0^L u(z)^2 dz \end{aligned}$$

Useful Tools in Infinite Dimension

These tools will be of major help in the following:

- the **variational derivative** of a functional: $\delta_X H$, in place of the gradient of the function, defined by

$$H(u + \varepsilon w) = H(u) + \varepsilon (\delta_u H, w)_{\mathcal{H}} + O(\varepsilon^2)$$

N.B. in the above easy example, $\delta_u H = u$.

- formally **skew symmetric operators**: $\mathcal{J}^T = -\mathcal{J}$, w.r.t the scalar product in the Hilbert space \mathcal{H} , i.e.

$$(u, \mathcal{J}v)_{\mathcal{H}} = -(\mathcal{J}u, v)_{\mathcal{H}}$$

- in the special case of quadratic Hamiltonian, hence linear dynamical systems $\dot{X} = \mathcal{A}X$: **semigroups**³.

³come on June, 4th, to ROMA Seminar by Ghislain Haine! 

Ex 3: Webster horn equations (1)

- the energy variables are: density ρ , and particle velocity v ,
- with pressure $p := c_0^2 \rho$, and energy density $U(\rho) := \frac{c_0^2}{2\rho_0} \rho^2$, let us define the Hamiltonian:

$$H(\rho, v) := \int_0^L \left(\frac{1}{2} \rho_0 v^2 + \rho_0 U(\rho) \right) S(z) dz,$$

- the co-energy variables are: $\delta_\rho H = \rho_0 U'(\rho) = \frac{1}{\rho_0} p$ and $\delta_v H = \rho_0 v$,
- then, the corresponding port-Hamiltonian system is given by

$$\frac{d}{dt} \begin{bmatrix} \rho \\ v \end{bmatrix} = \begin{bmatrix} 0 & -\frac{1}{S} \partial_z (S \cdot) \\ -\partial_z & 0 \end{bmatrix} \begin{bmatrix} \delta_\rho H \\ \delta_v H \end{bmatrix}. \quad (12)$$

- The wave equations which govern the acoustic pressure p and the particle velocity v are given by, respectively:

$$\frac{1}{c_0^2} \partial_t^2 p(z, t) - \frac{1}{S(z)} \partial_z [S(z) \partial_z p(z, t)] = 0, \quad (13)$$

$$\frac{1}{c_0^2} \partial_t^2 v(z, t) - \partial_z \left[\frac{1}{S(z)} \partial_z [S(z) v(z, t)] \right] = 0, \quad (14)$$

Ex 4: compressible Euler equations

Consider an **inviscid**, **irrotational** and **isentropic** fluid, in $\Omega \subset \mathbb{R}^3$:

$$\frac{d}{dt}\rho = -\operatorname{div}(\rho \mathbf{v}) \quad (15)$$

$$\frac{d}{dt}\mathbf{v} = -(\mathbf{v} \cdot \mathbf{grad})\mathbf{v} - \frac{1}{\rho}\mathbf{grad}p. \quad (16)$$

Following e.g. [van der Schaft & Maschke, 2001],

- Energy variables: ρ, \mathbf{v} ,
- Hamiltonian: $H(\rho, \mathbf{v}) := \int_{\Omega} \left(\frac{1}{2}\rho \mathbf{v} \cdot \mathbf{v} + \rho U(\rho) \right) dV$,
- Co-energy variables: $\delta_{\rho}H = \frac{1}{2}\mathbf{v} \cdot \mathbf{v} + h(\rho)$ and $\delta_{\mathbf{v}}H = \rho \mathbf{v}$,
- Dynamical system in standard form:

$$\frac{d}{dt} \begin{bmatrix} \rho \\ \mathbf{v} \end{bmatrix} = \begin{bmatrix} 0 & -\operatorname{div} \\ -\mathbf{grad} & 0 \end{bmatrix} \begin{bmatrix} \delta_{\rho}H \\ \delta_{\mathbf{v}}H \end{bmatrix};$$

Definitions for pHs (1)

Consider the dynamical system

$$\frac{d}{dt}X(z, t) = (\mathcal{J}) \delta_X \mathcal{H}(X) \quad (17)$$

with (quadratic) Hamiltonian :

$$\mathcal{H}(X) = \frac{1}{2} \int_0^L X(z, t)^T \mathcal{L}X(z, t) dz,$$

and (linear) variational derivative :

$$\delta_X \mathcal{H}(X) = \mathcal{L}X(z, t).$$

We suppose that operator \mathcal{J} is formally skew-symmetric.

Definitions for pHs (2)

The **energy balance** associated to this system is:

$$\begin{aligned} \frac{d\mathcal{H}}{dt}(t) &= \int_0^L \delta_X \mathcal{H}(X) \cdot \frac{dX}{dt} dz, \\ &= \int_0^L \delta_X \mathcal{H}(X) \cdot (\mathcal{J}) \cdot \delta_X \mathcal{H}(X) dz, \\ &= 0, \quad \text{since } \mathcal{J} \text{ is skew-adjoint? Almost!} \end{aligned}$$

For the horn equation, say with uniform cross-section $S(z) = S$,

$$\begin{aligned} \frac{d\mathcal{H}}{dt}(t) &= \int_0^L (-e_1 \partial_z e_2 - e_2 \partial_z e_1) dz, \\ &= \int_0^L -\partial_z (e_1 e_2) dz, \\ &= e_1(0) e_2(0) - e_1(L) e_2(L). \end{aligned}$$

\implies Energy flows through the **boundary**, only!

More involved examples in fluid mechanics

Some more specific Hamiltonian models for fluid mechanics are available for:

- **potential flow** (irrotational fluid), with the use of the **stream function**
- **incompressible** fluid (**very difficult**: an infinite-dimensional Differential Algebraic Equation!), with the constraint $\operatorname{div}(\mathbf{v}) = 0$, make use of the **vorticity vector** $\boldsymbol{\omega} := \operatorname{curl}(\mathbf{v})$.

⇒ read the book by [Olver, 1993]

⇒ come and talk with me, since I have read the book for you!

Definitions for pHs (1)

Consider the dynamical system

$$\frac{d}{dt}X(z, t) = (\mathcal{J} - \mathcal{R}) \delta_X \mathcal{H}_0(X) \quad (18)$$

with (quadratic) Hamiltonian :

$$\mathcal{H}_0(X) = \frac{1}{2} \int_0^L X(z, t)^T \mathcal{L}X(z, t) dz,$$

and (linear) variational derivative :

$$\delta_X \mathcal{H}_0(X) = \mathcal{L}X(z, t).$$

We suppose that

- operator \mathcal{J} is formally skew-symmetric,
- operator \mathcal{R} is positive self-adjoint.

Definitions for pHs (2)

The **energy balance** associated to this system is:

$$\begin{aligned}
 \frac{d\mathcal{H}_0}{dt}(t) &= \int_0^L \delta_X \mathcal{H}_0(X) \cdot \frac{dX}{dt} dz, \\
 &= \int_0^L \delta_X \mathcal{H}_0(X) \cdot (\mathcal{J} - \mathcal{R}) \cdot \delta_X \mathcal{H}_0(X) dz, \\
 &= - \int_0^L \delta_X \mathcal{H}_0(X) \cdot \mathcal{R} \cdot \delta_X \mathcal{H}_0(X) dz, \\
 &\leq 0.
 \end{aligned}$$

But when $\mathcal{R} \neq 0$, no underlying **Dirac structure** is to be found for this damped system, with efforts $e := \delta_X \mathcal{H}_0(X)$ and flows $f := \frac{dX}{dt}$, linked by $f = \mathcal{J} e$.

Ex 5: compressible Navier-Stokes

Consider an **irrotational** and **isentropic** fluid, in $\Omega \subset \mathbb{R}^3$:

$$\frac{d}{dt}\rho = -\operatorname{div}(\rho \mathbf{v}) \quad (19)$$

$$\frac{d}{dt}\mathbf{v} = -(\mathbf{v} \cdot \mathbf{grad})\mathbf{v} - \frac{1}{\rho} \mathbf{grad}p + \frac{1}{\operatorname{Re}} \Delta \mathbf{v}. \quad (20)$$

Still following [van der Schaft & Maschke, 2001],

- Energy variables: ρ, \mathbf{v} ,
- Hamiltonian: $H := \int_{\Omega} \left(\frac{1}{2} \rho \mathbf{v} \cdot \mathbf{v} + \rho U(\rho) \right) dV$,
- Co-energy variables: $\delta_{\rho} H = \frac{1}{2} \mathbf{v} \cdot \mathbf{v} + h(\rho)$ and $\delta_{\mathbf{v}} H = \rho \mathbf{v}$,
- Dynamical system in standard form:

$$\frac{d}{dt} \begin{bmatrix} \rho \\ \mathbf{v} \end{bmatrix} = \begin{bmatrix} 0 & -\operatorname{div} \\ -\mathbf{grad} & 0 \end{bmatrix} \begin{bmatrix} \delta_{\rho} H_0 \\ \delta_{\mathbf{v}} H_0 \end{bmatrix} - \begin{bmatrix} 0 & 0 \\ 0 & \mathcal{C} \end{bmatrix} \begin{bmatrix} \delta_{\rho} H \\ \delta_{\mathbf{v}} H \end{bmatrix};$$

Ex 5: compressible Navier-Stokes

With $\mathcal{C} = -\frac{1}{Re}\Delta$. It has the desired $(\mathcal{J} - \mathcal{R})$ form:

- \mathcal{J} is a skew-symmetric operator, since the formal adjoint of div is $-\mathit{grad}$,
- \mathcal{R} is a symmetric and positive operator, since $-\Delta$ is.

\implies more important, the parametrization $\mathcal{R} = \mathcal{G}\mathcal{S}\mathcal{G}^*$ is very easily found to be:

$$\mathcal{G} := \begin{bmatrix} 0 \\ \mathit{grad} \end{bmatrix}, \quad \mathcal{G}^* = [0 \quad -div], \quad \text{and} \quad \mathcal{S} := \frac{1}{Re} I.$$

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and now? the show must go on!

Let us welcome: Flávio Ribeiro!